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DG1146/17.55 - II13 型锅炉防结焦优化调整

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摘 要: 某厂1 号炉为 330 MW 亚临界四角切圆锅炉,采用 低氮燃烧器,燃烧器分层布置,上下层间距2 m。入炉煤为 准东高钠煤,锅炉满负荷运行时,炉内火焰充满度较高,温度 分布与速度分布呈"蝴蝶型",且上层主燃区 CO 浓度较高, 还原性气氛较强,锅炉上层主燃区出现严重结焦现象。针对 这一问题,采用改变上层燃烧器一、二次风配风方式对炉内 燃烧进行优化调整。结合温度场试验数据与 FLUENT(流体 动力学)数值模拟的方法,研究了优化前后炉内风速、温度 及组份的分布特性。研究表明:燃烧优化调整后上层燃烧器 区域 CO 浓度由 0.8%降低至 0.5%,且上层燃烧器区域风速 分布与温度分布相对收缩,抑制结焦效果显著。

关 键 词: 准东高钠煤; 四角切圆锅炉; 分层布置; 结焦; 数 值模拟; 蝴蝶型

中图分类号: TK229.1 文献标识码: A DOI:10.16146/j.cnki.rndlgc.2015.06.020 引言

中国新疆准东高钠煤中钠存在形式主要以水溶 钠为主,煤粉颗粒中孔隙结构越丰富则水溶钠含量 越高,导致高钠煤中钠的总量也越多。煤中存在的 钠直接关系到煤的灰污性质,以高含量钠煤为燃料 的锅炉均会出现结渣、沾污、积灰和腐蚀等问题,严 重影响了锅炉的正常运行^[1]。

某厂1 号炉为 330 MW 亚临界四角切圆锅炉, 采用低氮燃烧器,燃烧器分层布置,上下层间距 2 m。入炉煤为准东高钠煤,修前锅炉满负荷运行 下,锅炉上层主燃区出现严重结焦现象。本研究通 过改变燃烧上层一、二次风配风方式,结合温度场试 验与 FLUENT 数值模拟的方法,研究了优化前后炉 内速度、温度及组份的分布特性,结果表明:优化后 炉内高温区集中在主燃区内部,速度场与温度场收 缩,烟气刷墙程度减弱,同层燃烧器截面烟温偏差减 小,上层主燃区还原性气氛减弱,炉内未出现严重结 焦情况,证明了优化方案的正确性。 1 研究对象

某电厂1号锅炉为 DG1146/17.55 – II13 型自 然循环单炉膛 π 型布置一次中间再热、平衡通风、 固态排渣的 330 MW 亚临界四角切圆燃烧锅炉。该 炉燃烧器采用分层布置方式,上6 层下7 层布置。 锅炉结构及燃烧器布置如图1 所示。煤粉细度 R₉₀ =22%(设计煤种)。采用水平浓淡燃烧器,四角布 置,切圆燃烧,所有喷口均可上下摆动 30°。炉膛宽 度为14 706.6 mm,深度为13 743.4 mm,燃烧器上 一次风喷口到大屏过热器底部距离为19 280 mm, 燃烧器下一次风喷口到水冷壁冷灰斗拐角之间距离 为4 023 mm。

2 炉内燃烧模拟

2.1 数学模型与网格划分

数值模拟采用 FLUENT 软件。采用分段方式对 炉膛进行网格划分,从冷灰斗至炉膛出口划分为 8 个部分。炉膛截面网格采用 Paving 方法生成非结 构四边形网格,体网格用 Cooper 方法沿着炉膛高度 方向铺展生成六面体网格。用 Paving 方法生成的 辐射状网格线与四角射流的气流轨迹基本平行,网 格线与流线的夹角进一步减小,并降低了数值伪扩 散。为保证燃烧器区域计算精度,对该区域网格加 密。进行网格独立性验证之后,整个炉膛网格总数 划分为 71 万个^[2-3]。

主燃烧器喷口均采用速度入口边界条件,假设 煤粉温度和进风温度恒定,炉膛本体保持恒温;炉膛 出口采用压力出口边界条件;煤粉粒径满足 Rosin – Rammlar 分布。利用 FLUENT 软件进行数值模拟, 气相湍流流动采用标准 κ – ε 双方程模型,焦炭燃

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烧采用动力学/扩散控制反应速率模型 辐射传热采 用 P1 辐射模型 煤粉挥发分的热解采用双步竞争反 应模型 煤粉颗粒跟踪采用随机轨道模型 ,气相湍流 燃烧采用混合分数 – 概率密度函数模型 ,压力速度 耦合采用 SIMPLE 算法^[4]。



(a) 锅炉结构简图



(b) 上层燃烧器喷口布置简图(mm)



(c) 下层燃烧器喷口布置简图(mm)

图1 锅炉结构及燃烧器布置

Fig. 1 Boiler structures and burner arrangement





(b) 主燃烧器区域横截面网格

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(c) 主燃烧器区域深度方向截面网格

图 2 炉膛网格划分

Fig. 2 Furnace grid meshing

2.2 边界条件

将燃烧器喷口定义为速度入口,并给定速度与 温度;炉膛出口为压力出口,并设定出口压力为-60 Pa;固体壁面采用无质量渗透且无速度滑移边界条 件,同时使用标准壁面函数来考虑对壁面的影响。

速度入口水力直径 *D*_h、湍流强度 *I* 按下式 计算: $D_{\rm h} = 4A/d$ mm $I = 0.16 (R_{\rm e})^{-0.125}$ %

式中: *d* — 接触面周长, mm; *Re*—入口处雷诺数^[5-7]。

2.3 工况设置

优化方案为调整上层燃烧器一、二次风配风方 式。表1为入炉煤质分析 ,表2给出优化前后满负 荷运行配风方式主要相关参数。研究工况为锅炉满 负荷运行工况。

表1 准东高钠煤煤质分析(%)

Tab. 1 Coal quality analysis of high sodium coal

| 符号 | 设计煤种 | 燃用煤种 |
|---|-------|-------|
| C _{ar} | 58.4 | 58.58 |
| H_{ar} | 3.86 | 2.86 |
| O_{ar} | 8.48 | 11.35 |
| $\mathbf{N}_{\mathbf{ar}}$ | 0.91 | 0.47 |
| S_{tar} | 0.68 | 0.49 |
| M_{t} | 8.7 | 20.2 |
| $A_{ m ar}$ | 18.94 | 6.05 |
| $Q_{ m net. \ ar}/ m kJ$ • kg ⁻¹ | 22100 | 20920 |

表 2 满负荷运行配风方式

Tab. 2 Running at full capacity with the way the wind

| 项目 | 优化前风速/m・s⁻¹ | 优化后风速/m・s ⁻¹ | 风温/℃ |
|-----|-------------|-------------------------|------|
| 一次风 | 28 | 34 | 75 |
| 二次风 | 49 | 55 | 337 |

表 3 为燃用设计煤种时 BMCR(锅炉最大连续 蒸发量工况)、BRL(额定工况)下炉膛出口烟温。

表 3 炉膛出口与高温过热器出口烟气温度设计值

Tab. 3 Furnace exit flue gas temperature design value

| 工 况 | BMCR | BRL |
|-----------|------|------|
| 炉膛出口烟温/℃ | 1141 | 1127 |
| 高温过热器出口/℃ | 746 | 741 |

3 数值模拟结果及分析

3.1 速度分布

图 3 为优化前后炉膛中心纵截面速度分布。可 以看出,炉内最高风速区出现在主燃区内,速度场呈 "蝴蝶型"分布,在下层燃烧器至上层燃烧器的整个 主燃区内部 速度场呈"下窄上宽"逐渐充满的发展 趋势,上层燃烧器区域充满度最高。优化后上层燃 烧器区域速度场收缩,炉膛充满度相对减小,上层主 燃区射流刚性增强,携带煤粉颗粒的穿透力增强,煤 粉颗粒着火位置向炉膛中心靠拢。



(a) 优化前



(b) 优化后

图 3 炉膛中心纵截面速度场分布

Fig. 3 Furnace center longitudinal section velocity field

图 4 - 图 6 分别为优化前后炉内上、下层燃烧

器不同横截面速度分布。由图 4 可以看出,优化前 后下层燃烧器区域速度分布大致相同,且形成较好 切圆,未出现射流刷墙情况。一次风射流刚性较强, 射程较大,射流在喷口附近的衰减速度比射流中后 段小。由于射流的卷吸作用,射流流动方向上的流 量逐渐增减,射流宽度也随着增加,有利于煤粉与助 燃空气的混合,而射流卷吸周围的高温烟气,有利于 煤粉的着火和稳定燃烧^[8-10]。



(a) 优化前



(b) 优化后

图4 A 层一次风喷口横截面速度分布

Fig. 4 A cross – sectional layer of the primary air jet velocity distribution

由图 5 可以看出,由于上、下层燃烧器的卷吸作 用,中部缓冲区形成较好的切圆,速度分布结构较之 上、下层燃烧器相对简单。最高风速区并未在炉膛 四角处,而是在假想切圆附近。上、下两层燃烧器之 间,虽有中部缓冲区隔离,但仍存在相互影响。由图 6 可以看出,优化后上层燃烧器 D 层速度场明显收 缩,气流刷墙情况明显减少,这是由于优化燃烧之后 上层燃烧器射流刚性增强所致。









Fig. 5 Central buffer cross-sectional velocity distribution

3.2 温度场分布

图 7 为炉膛中心纵截面温度分布。由图 7 可以 看出 整个炉膛高温区出现在主燃区内部。随炉膛 高度的增加 ,温度水平呈逐渐降低趋势。在燃烧器 区域 ,由于煤粉气流在射出喷口时 A 股射流相互撞 击 在炉膛中央形成一个强烈旋转的旋涡 ,它延长了 煤粉颗粒在高温环境下的行程和停留时间 ,有利于 煤粉的充分燃尽。在主燃区内 ,随着炉膛高度的增 大 ,火焰的炉膛充满度呈渐增趋势 ,炉内温度同炉内 速度分布相似 ,亦呈 "下窄上宽"分布 ,且在主燃烧 区域形成 "蝴蝶型"高温区域; 优化前上层燃烧器区 域 ,高温区贴近壁面 ,且温度分布不均。优化后炉内 温度分布均匀,上层燃烧器区域火焰中心向炉膛中 心收缩,延长了煤粉颗粒的着火时间。高温区远离 燃烧器附近壁面区域,避免燃烧器区域壁面热负荷 过高,降低该区域结焦产生的可能。下层主燃区高 温区分布都较对称,收缩较小,且火焰中心位置未出 现抬高或降低情况,炉膛出口处烟气温度变化不大, 这说明优化后煤粉颗粒在上层主燃区内迅速燃尽, 未延长煤粉颗粒的炉内燃尽时间。



(a) 优化前





Fig. 6 D cross-sectional layer of the primary air nozzle velocity distribution

图8 和图9 分别为下层燃烧器 A 层与上层燃烧器 D 层的一次风喷口横截面温度分布。优化后 A 层燃烧器横截面温度场分布较优化前变化不大,主要原因是分层燃烧之后,上下层燃烧器之间具有相互独立性,改变上层燃烧器配风方式,对下层主燃区影响有限。优化前 D 层温度分布不均,高温区贴近炉墙。优化后 D 层温度场明显收缩,温度分布较优化前均匀,烟温偏差减小,降低了燃烧器区域局部热负荷。



图7 炉膛中心纵截面温度分布



3.3 组份分布

图 10 和图 11 分别为炉膛横截面平均氧浓度与 CO 浓度随炉膛高度变化曲线。由图 10 可以看出, 主燃区分层布置,在上下层主燃区中间有氧浓度较 低的中部缓冲区。优化前后,下层燃烧器配风方式 不变,故此区域氧浓度在优化前后基本一致。在20 m处由于燃尽风的补入,氧浓度迅速增大。优化后, 上层主燃区风量增大,氧浓度较优化前明显提高,氧 化性气氛增强,起到抑制结焦的作用。



(a) 优化前



(b) 优化后



由图 11 可以看出,优化前后下层主燃区 CO 浓度变化趋势基本相同,优化后下层主燃区 CO 浓度略高。这是由于上层主燃区配风方式改变后,风量的增大 影响炉膛温度水平,不利于可燃物燃尽。经过燃尽风区,CO 浓度明显降低,优化前后 CO 浓度基本一致。因此风量增大对于可燃物燃尽的影响很小,可以忽略。优化后上层主燃区的 CO 浓度明显较优化前低,还原性气氛较弱,且优化后上层主燃区速度场与温度场收缩,火焰相对远离燃烧器区域,故上层主燃区的结焦抑制能力显著增强。



图 9 D 层一次风喷口横截面温度分布 Fig. 9 D cross-sectional layer of the primary air nozzle temperature distribution











3.4 温度场试验

由于该锅炉所燃用煤种为准东高钠煤,其挥发 分、水分含量较高,灰分含量较低,灰熔点较低,一般 为1200℃左右。判断燃煤结焦特性常用灰软化温 度*ST*(℃)作为判据。*ST*判据的判别界限如表4所 示。根据表4中*ST*判据标准,该锅炉燃用准东煤为 极易结焦煤种,易造成炉内出现结焦现象。

表 4 结焦特性等级判别界限(ST 判据)

| STE 190 | 往沐柱州 |
|-------------|------|
| 517 C | 结進特性 |
| > 1480 | 低 |
| 1480 - 1370 | 中 |
| 1370 - 1270 | 高 |
| < 1270 | 严重 |

采用镭射高温测试仪测量炉内温度。图 12 为 优化前 E 层看火孔结焦情况 ,表 5 为优化前炉内温 度场试验数据。

由表 5 可以看出,炉内同层横截面烟温分布不 均,炉膛四墙区域温差最高达 270 ℃左右,表明炉内 温度场较紊乱,与模拟温度场结果吻合。结合图 13 和表 5 可以看出,上层主燃区 E 层燃烧器附近 4 个 角看火孔处均被炉内焦渣堵塞,导致无法测试此处 火焰温度。运行所燃用煤种为挥发分高且灰熔点较 低的准东煤。由模拟结果知,锅炉运行中上层主燃 区火焰刷墙 .CO 浓度较高 ,壁面附近有较强的还原 性气氛 ,烟气中灰熔点进一步降低 ,因此造成了炉膛 的严重结焦。



图 12 优化前 E 层看火孔结焦情况 Fig. 12 Coke case at the fire observation holes of E-layer before adjustment

表5 优化前温度场试验数据(℃)

Tab. 5 Furance temperature at the fire observation holes of layers before adjustmen(°C)

| 位置 | 1 号角 | 2 号角 | 3 号角 | 4 号角 |
|---------|-------|-------|-------|-------|
| E 层燃烧器 | 结焦 | 结焦 | 结焦 | 结焦 |
| D 层燃烧器 | 结焦 | 1 323 | 1 177 | 结焦 |
| C 层燃烧器 | 1 358 | 1 189 | 1 232 | 1 085 |
| B 层燃烧器 | 1 191 | 1 079 | 1 195 | 1 045 |
| A 层燃烧器 | 1 044 | 1 122 | 1 218 | 1 194 |
| E 层烟温偏差 | / | / | / | / |
| D 层烟温偏差 | / | 73 | -73 | / |
| C 层烟温偏差 | 142 | - 27 | 16 | - 131 |
| B 层烟温偏差 | 22 | - 48 | 68 | - 82 |
| A 层烟温偏差 | -41 | -23 | 73 | 49 |



图 13 优化后 E 层看火孔结焦情况 Fig. 13 Coke case at the fire observation holes of E-layer after adjustmen

表6 优化后温度场试验数据(℃)

Tab. 6 Furance temperature at the fire observation

holes of layers after adjustmen($^{\circ}C$)

| 位置 | 1 号角 | 2 号角 | 3 号角 | 4 号角 |
|---------|-------|-------|-------|-------|
| E 层燃烧器 | 1 370 | 结焦 | 1 395 | 1 309 |
| D 层燃烧器 | 1 275 | 结焦 | 1 248 | 1 294 |
| C 层燃烧器 | 1 158 | 1 246 | 1 194 | 1 258 |
| B 层燃烧器 | 1 214 | 1 184 | 1 158 | 1 080 |
| A 层燃烧器 | 1 174 | 1 149 | 1 187 | 1 146 |
| E 层烟温偏差 | 12 | / | 37 | - 49 |
| D 层烟温偏差 | 2 | / | - 25 | 21 |
| C 层烟温偏差 | - 36 | 52 | 0 | 64 |
| B 层烟温偏差 | 80 | 50 | 24 | - 54 |
| A 层烟温偏差 | 10 | - 15 | 23 | - 18 |

由表6可以看出,优化后燃烧器各层截面温度 分布较均匀,烟温偏差较小,结焦程度大幅降低。这 说明优化后,炉内速度分布与温度分布较优化前均 匀,燃烧组织良好,炉内未出现严重射流刷墙与局部 热负荷过高的情况,且炉内还原性气氛减弱,很大程 度上抑制了炉内结焦的产生。

4 结 论

(1)该炉燃烧器分层布置,炉内形成较为对称的"蝴蝶型"速度场与温度场,且呈"下窄上宽" 分布。

(2)燃烧调整前上层主燃烧区域烟气刷墙,壁 面热负荷过高,同一截面烟温分布不均,炉膛四墙区 域烟温偏差达270 ℃以上,入炉煤为灰熔点较低的 准东高钠煤,且上层主燃区有较强的还原性气氛,政 使锅炉燃烧器 D、E 层等上层主燃烧区域结交严重。

(3) 通过燃烧优化调整,提高 D、E 层燃烧器一次风速至 34 m/s,二次风速至 55 m/s,上层主燃区 射流刚性增强,炉内速度场与温度场收缩,煤粉颗粒 着火时间延长。

(4) 燃烧优化调整后,上层主燃烧区域 CO 浓度由 0.8% 降至 0.5% 浓度范围,还原性气氛相对减弱,上层主燃区的结焦起到了抑制作用。

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(丛 敏 编辑)

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In the light of characteristics of faults in milling systems in power plants and current problems existing in the study in the domain a fault diagnostic method based on the mean impact value algorithm and a probability neural network was proposed. Firstly the training samples were pretreated to eliminate any abnormal samples and enlarge the learning scope of the neural network. Secondly by making use of the mean impact value algorithm the impact values of various fault sign parameters on the fault types were calculated and listed out in order to choose the main parameter influencing the diagnostic results and achieve the aim of simplifying the attributes. On this basis the simplification results were used as the inputs to the probabilistic neural network. Finally the fault samples were input into the probabilistic neural network to conduct training and testing to obtain the simulation results. The case diagnostic results show that the method in question can expand and increase the kinds of faults identified shorten the time duration for diagnosis and enhance the diagnostic correctness rate. **Key words**: mean impact value algorithm probabilistic neural network fault diagnosis milling system

DG1146/17.55-II13 型锅炉防结焦优化调整 = Anti-coking Optimization and Adjustment of a DG1146/17.

55-**H**13 **Type Boiler** [刊 說】]LU Tai ,HE Pei-ye (College of Energy Source and Power Engineering ,Northeast University of Electric Power ,Jilin ,China ,Post Code: 132012) ,XU He (Technology and Information Center ,CPI Henan Electric Power Co. Ltd. Zhengzhou ,China ,Post Code: 450001) //Journal of Engineering for Thermal Energy & Power. - 2015 ,30(6). -903 -910

Boiler No. 1 in a power plant is a 330 MW subcritical tangentially-fired one ,which adopts low nitrogen burners and a layered arrangement of burners with the spacing between the upper and lower layers being 2 m. The coal fed into the boiler is Zhundong-originated coal with a high sodium content. When the boiler is operating at its full load ,the filling degree of the flame in the furnace is relatively high ,the temperature and speed distribution assume a "butterfly shape" ,the concentration of carbon monoxide in the main combustion zone in the upper layer is comparatively high and the reduction action ability is relatively strong ,however ,a serious coking phenomenon occurs in the main combustion zone in the upper layer of the boiler. To solve this problem ,the primary and secondary air distribution mode of the burners in the upper layer was changed to optimize and make an adjustment of the combustion in the furnace. According to the test data of the temperature field and by making use of the numerical simulation method provided in the software Fluent the distribution characteristics of the temperature speed and various constituents inside the furnace before and after the optimization were studied. It has been found that after the optimization and adjustment of the combustion inside the furnace the carbon monoxide concentration in the zone of the burners in the upper layer decreases from 0.8% to 0.5% and the speed and temperature distribution region correspondingly reduces thus making an outstanding achievement in containing the coking. The method above-mentioned can offer reference for anti-coking operation and modification of boilers burning Zhundong-originated coal. **Key words**: Zhundong-originated coal with a high sodium content tangentially-fired boiler layered arrangement coking numerical simulation butterfly shape

多流量低温省煤器最优设计通用数学模型 = A General-purposed Mathematical Model for Optimal Design of Multi-flow Low Temperature Economizers [刊,汉]TAN Liang-hong, HU San-gao, ZHAO Yan, WANG Zhe (College of Energy Source Power and Mechanical Engineering, North China University of Electric Power, Beijing, China Post Code: 102206) //Journal of Engineering for Thermal Energy & Power. - 2015 30(6). -911-915

For low temperature economizers in boilers proposed was a multi-flow connection mode ρ f which the connection criteria and the resulting influence on the cost-effectiveness of the unit were analyzed. Under the condition of the total investment power consumption and amount of coal burned being taken into account and with the maximal pure income serving as the object of study a mathematical model for optimal design of low temperature economizers was established and with a N200-12.75/535/535 unit serving as an example ρ calculation was performed. The calculation results show that compared with a single-flow economizer ρ multi-flow economizer can increase the amount of steam extracted at low parameters and at the same time decrease that at high parameters thus better realizing a stepped utilization of waste heat from flue gases reducing the coal consumption by 0.5 g/(Kw. h) cutting the heat exchange area by 1 000 square meters and minimizing the initial investment and the power consumed by the fans and pumps. **Key words**: utility boiler low temperature economizer ρ ptimized design

热电联产机组协调控制系统优化设计 = Optimized Design of a Coordinated Control System for Cogeneration Units [刊,汉]GUO Xiao-hong, CHEN Qi (Inner Mongolia Electric Power Academy, Hohhot, China, Post Code: 010020), TIAN Liang (College of Control Science and Engineering, North China University of Electric Pow-