热力工程

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流化床反应器内气固两相流动特性的研究

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摘 要:基于颗粒动力学理论模拟颗粒相流动,应用流体与 颗粒两相流理论考虑两相相间作用,建立了流化床核反应器 内多相流流动的计算流体动力学模型,数值模拟研究了流化 床核反应器内的流体动力行为。计算结果表明,应用 Gi dasPow曳力模型得到的沿截面颗粒浓度分布与已有实验结 果的分布趋势比较接近。在中心喷射区的中心处颗粒浓度 较高。随着径向距离的增大,逐渐降低到局部最小值后颗粒 浓度逐渐上升。在环隙区域内颗粒浓度基本保持不变。分 析了流体与颗粒间作用力、颗粒弹性恢复系数等对流化床核 反应器内流体动力特性的影响。研究表明,颗粒碰撞恢复系 数越大,流场内沿截面颗粒浓度分布变得越均匀。

关 键 词: 流化床核反应器; 两相流理论; 数值模拟 中图分类号: ^{TK2}24.1 文献标识码: A

引 言

流态化高温气冷堆具有非能动安全特性,主要 特征是采用流态化技术和高温气冷堆技术,实现均 匀的颗粒浓度、气相和燃料颗粒温度分布,提高传热 能力^[1]。目前,国际上已有关于流化床核反应器内 气固两相运动特性研究的报道,但发表的文献很少。 其中采用流化床核反应器在床层被流化时,床内存 在大气泡和腾涌等流化恶化的现象^[2],这在很大程 度上影响了反应器内核燃料颗粒的裂变反应率、堆 内能量输出特性以及堆芯安全系数。而在国内关于 带有倒锥体核反应器数值模拟领域的研究还未见报 道。本研究采用的带有倒锥体流化床核反应器,优 点在于此种结构的流化床核反应器具有较高的对流 换热系数和较大的热传导表面积以及较好的混合特 性。由于在流化过程中气体和颗粒之间混合均匀, 提高了气固两相的接触效率,从而增加了传热能力, 避免堆芯烧毁。

1 数学模型

流化床核反应器内流体动力学模型满足质量守 恒、动量守恒和能量守恒原理。同时为了简化流化 床核反应器内气体和颗粒流动计算,假设:(1)燃料 颗粒为球形,直径为常数;(2)气体和燃料颗粒无反 应,流动为等温流动过程。对于等温气固两相流动 过程,连续性方程可以表示为(i= ^g时为气体,i= ^s 时为颗粒相);

式中: ᠙⊢ 相密度, ^{kg/m³}; ε⊢ 相体积浓度; ╙- i 相速度, ^{m/ §}

气相动量守恒方程需要考虑气体与燃料球颗粒 之间的相互作用,可表示为:

$$\frac{\partial}{\partial t} (\epsilon_{g} \rho_{g} u_{g}) + \nabla (\epsilon_{g} \rho_{g} u_{g}) = -\epsilon_{g} \nabla P + \epsilon_{g} \rho_{g} g + \beta(u_{g} - u_{g}) + \nabla \cdot \tau_{g}$$
(2)
式中: ^g-重力加速度, ^m/ ^s P-气相压力, Pa β-气
固两相间的曳力系数; ^μ-气相动力粘度, Pa s
 τ_{g} -气相应力张量:

$$\tau_{g} = \mu_{g} \left[\nabla \mathfrak{l}_{g} + \left(\nabla \mathfrak{l}_{g} \right)^{\mathrm{T}} \right] - \frac{2}{3} \mu_{g} \left(\nabla \circ \mathfrak{l}_{g} \right) \mathrm{I} (3)$$

同理,颗粒相动量守恒方程除了需要考虑气体 与颗粒之间的相互作用外,还需要考虑颗粒相互碰 撞产生的作用力:

$$\frac{\partial}{\partial t} (\varepsilon_{s} \rho_{s} u_{s}) + \nabla \circ (\varepsilon_{s} \rho_{s} u_{s} u_{s}) = -\varepsilon_{s} \nabla P + \beta \times$$

$$(u_{s} - u_{s}) + \nabla \circ \tau_{s} + \varepsilon_{s} \rho_{s} g$$

$$(4)$$

式中: ^τ^s一固相应力张量, 由颗粒动理学方法固相应 力可表示为^[3~4]:

 $\tau_s = (-P_s + \lambda_s \nabla \cdot u_s) + 2\mu_s S_{[[} \nabla u_s +$

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$$\left(\nabla \mathbf{u}_{s}\right)^{\mathrm{T}} = \frac{2}{3} \left(\nabla \cdot \mathbf{u}_{s}\right) \mathbf{I}$$

$$(5)$$

式中: P_s--颗粒相压力, P^a; λ_s--颗粒相表观粘度, μ_s--颗粒相动力粘度, P^{a。}; P_s和 μ_s由动力 摩擦 应力模型确定,它们分别为:

$$P_{s} = \varepsilon_{s} \rho_{s} [1 + 2(1 + \Theta \varepsilon_{s} g] \Theta + Fr \frac{(\varepsilon_{s} - \varepsilon_{s} m_{in})^{n}}{\varepsilon_{s max} - \varepsilon_{s})^{p}}$$
(6)

$$\lambda_{s} = \frac{4}{3} \varepsilon_{s} \rho_{s} d_{s} g (1 + e) \sqrt{\frac{\Theta}{\pi}}$$
(7)

$$\mu_{s} = \frac{4}{5} \varepsilon_{s}^{2} \rho_{s} d_{s} g \left(1 + \mathfrak{S} \sqrt{\frac{\Theta}{\pi}} + \frac{10 \rho_{s} d \sqrt{\Theta \pi}}{96 \left(1 + \mathfrak{S} \frac{\Theta}{\mathfrak{S}}\right)} \times 1 + \frac{4}{5} \varepsilon_{s} \left(1 + \mathfrak{S} \right)^{2} + F \frac{(\varepsilon_{s} - \varepsilon_{s} \min)^{n} sip}{(\varepsilon_{s\max} - \varepsilon_{s})^{p} \sqrt{\frac{1}{2}}}$$
(8)

式中: $\varepsilon_{s,max}, \varepsilon_{s,min}$ -填充颗粒浓度和临界颗粒浓度; Θ -颗粒温度, $(m/s)^2, \Theta = d/3, u'$ -颗粒的脉动速度, m/s P P和 F-与颗粒材料物性有关的经验系数, 对于玻璃珠, 颗粒参数 P F 分别为 2.0 5.0 和 0.05^[5]; ϕ -内摩擦角, $(°), \phi$ 为 28.5°; J_D -应变率张量第二不变偏量。

颗粒温度 ⁽¹⁾ 可按固相脉动能量守恒方程确定:

 $\frac{3}{2} \begin{bmatrix} \frac{\partial}{\partial t} (\varepsilon_{s} \rho_{s} \Theta) + \nabla \cdot & (\varepsilon_{s} \rho_{s} \Theta \ u_{s}) \end{bmatrix} = (-\nabla P_{s} H + \tau_{s}) \cdot \nabla u_{s} + \nabla \cdot & (k \nabla \Theta) q_{s} - \gamma_{s} + \phi_{s}$ (9)

式中: +单位向量; k—颗粒相热传导系数; γ,—颗 粒脉动能耗散率; •,—气体与颗粒间脉动能交换, 它 们分别为:

$$k = \frac{75\rho_{s} d^{s} \sqrt{\pi \Theta}}{192 (1+e) g} \left[1 + \frac{6}{5} (1+e) g_{\varepsilon} \right]^{2} + 2\varepsilon_{s}^{2}\rho_{s} (1+e) \sqrt{\frac{\Theta}{\pi}}$$
(10)

$$\gamma_{s} = 3(1-\frac{e}{2})\varepsilon_{s}^{2}\rho_{s}g\Theta\left(\frac{4}{d}\sqrt{\frac{\Theta}{\pi}}-\nabla \cdot \mathbf{u}\right) \quad (11)$$

$$\phi_{s} = -3\beta\Theta \qquad (12)$$

式中: e-颗粒非弹性碰撞恢复系数; d--颗粒直 径, m, g--颗粒径向分布函数:

$$g = \frac{3}{5} \left[1 - \left(\frac{\varepsilon_s}{\varepsilon_{sma}} \right)^{1/3} \right]^{-1}$$
(13)

由于颗粒的离散特性,对于颗粒群的受力进行 理论分析非常困难,通常颗粒的滞止阻力较其它受 力更为重要,对颗粒的运动起支配作用。在气固系 统中,实验观测到由于邻近颗粒的存在,颗粒的阻力 较单颗粒的阻力要大,因此在计算稠密气固两相流 时需要考虑邻近颗粒的影响,即对阻力系数进行修 正,将其与颗粒雷诺数以及颗粒体积分数进行关联。 对阻力系数的修正主要来自实验的数据,其中阻力 系数修正式为:

(1) 基于 E^{rgun}方程的阻力公式适用于稠密的 固定颗粒床:

$$F_{D} = \frac{V_{s\beta}}{1 - \varepsilon_{g}} (u_{g} - u_{s})$$
(14)

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$$\beta = 150 \frac{\varepsilon_s^2 \mu}{\varepsilon_g (\phi_s d)^2} + 1.75 \varepsilon_s \frac{\rho_g}{\phi_s d} | u_g - u_g | \quad (15)$$

(2) W^{en}& Y^u的指数修正公式适用于较稀疏 的两相流:

$$\beta = \frac{3}{4} C_{\rm D} \frac{\varepsilon_{\rm s} \rho_{\rm g} | \mathbf{u}_{\rm g} - \mathbf{u}_{\rm s} |}{d_{\rm s}} \varepsilon_{\rm g}^{-2.65}$$
(16)
$$C_{\rm D} = \begin{cases} \frac{24}{R} e^{(1+0.15R^{\oplus.687})} & \text{R} \leq 1000 \end{cases}$$

(3) Gilaspow模型^[6],在稀相区和密相区分别 采用 Wen& Yu公式以及 Ergun方程:

$$\beta = \begin{cases} 150 \frac{\varepsilon_{s}^{2} \mu}{\varepsilon_{g} (\phi_{s} d_{s})^{2}} + 1.75 \varepsilon_{s} \frac{\rho_{g}}{\phi_{s} d_{s}} | u_{g} - u_{s} | & \varepsilon_{g} \leq 0.8 \\ \frac{3}{4} C_{D} \frac{\varepsilon_{s} \rho_{g} | u_{g} - u_{s} |}{d_{s}} \varepsilon_{g}^{-2.65} & \varepsilon_{g} > 0.8 \end{cases}$$

$$(18)$$

(4) A rastoop our left
$$\vec{z}^{[7]}$$
:

$$\beta = \frac{1}{Re} + 0 \quad 336 \quad d_{g} \mid u_{g} - u_{g} \mid \varepsilon_{s}\varepsilon_{g} \quad (19)$$
(5) DiFelicelle T $\pi^{[8]}$.

$$\mathbf{f}(\boldsymbol{\varepsilon}_{g}) = \boldsymbol{\varepsilon}_{g}^{\alpha} \tag{20}$$

$$\alpha = 4.7 - 0.65 \exp\left[-\frac{(1.5 - \log_{70} R^{\circ})^2}{2}\right]$$
 (21)

(6) Syam [a lO' Brien 阻力模型^{[9}:

$$\beta = \frac{3}{4} C_{\rm D} \frac{\varepsilon_{\rm g} \varepsilon_{\rm s} \rho_{\rm g}}{V_{\rm r}^2 d} | \mathbf{u}_{\rm g} - \mathbf{u}_{\rm s} | \qquad (22)$$

$$C_{\rm D} = (0 \ 63 + 4 \ 8 \ \sqrt{\frac{V_{\rm r}}{R}})^2$$
 (23)

$$V_r = \frac{1}{2} [a - 0.06 \text{ Re} +$$

$$\sqrt{(0\ 06\ R^{6})^{2} + 0\ 12\ R^{6}(2\ b-a) + a^{2}}]$$
(24)
$$a - e^{\frac{4}{3}14}$$
(25)

$$b = \begin{cases} 0 & 8\varepsilon_g^{L,28} & \varepsilon \leq 0 & 85 \\ \varepsilon_g^{2,65} & \varepsilon > 0 & 85 \end{cases}$$
(26)

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2 模拟结果和分析

在气相入口处,给定入口轴向气体速度 ^u,径 向速度为零;在壁面处采用无滑移条件;出口条件, 取充分发展的管流条件 $\frac{\partial \phi}{\partial x} = 0$ ($\phi = \mu$ ^v)。近壁面处 采用壁面 函数 处理 方法。 ^u方 程: $\tau_{wall} = C_f \frac{1}{2}$ $\varepsilon_{g}\rho_{g}$ ^d; ^v方程: $\tau_{wall} = C_f \frac{1}{2} \varepsilon_{g}\rho_{g}\nu^{2}$ 。其中, $C_f = 0.003$ 为摩擦系数。

対颗粒相,取入口颗粒速度为零;在壁面处颗粒 为无滑移条件, $u_s = v_s = 0$,出口条件,取充分发展的 管流条件, $\frac{\partial}{\partial r} = 0$ ($\phi_s = \rho, u, v_s, \Theta$)。

采用带有倒锥体的流化床核反应器结构及反应 器内流动如图 1所示,反应堆堆芯内存储一定量的 燃料球颗粒,氦气作为冷却剂。气体由反应堆底部 进入,流过燃料球颗粒,形成高温气体,最后由反 应器顶部送出。反应器的几何尺寸借鉴 Pain等人 的流化床核反应器结构^[1],在模拟的反应器内假设 流动为轴对称。 Pain等人采用的冷却流体为氦 气^[1],流化床核反应器的运行压力为 6 MPa模拟计 算时的冷却氦气密度为 0 162 5 k^{g/m³},核燃料颗粒 直径和颗粒密度分别为 25 mm和 1 920 k^{g/m³}。



图 1 流化床核反应器堆芯结构



图 2 颗粒浓度标准方差的径向分布

图 2为入口射流速度 30^{m/}时采用不同曳力 流场各部分的颗粒浓度 ε_s的标准方差 σ 来定量衡 模型床内颗粒浓度的标准方差分布。直观地可以用 量计算域内颗粒分布的不均匀程度 (其中 σ = ?1994-2018 China Academic Journal Electronic Publishing House. All rights reserved. http://www.cnki.net

 $\sum_{i=1}^{n} (\varepsilon_i + \varepsilon_{av})^2 / N, \varepsilon_{av}$ 为平均颗粒浓度, N为计算 样本数)。采用 Anaspopout模型、Syam la lo Brien模 型以及 DiFelice模型进行模拟的结果表明,在喷泉 高度位置以下,颗粒浓度的标准方差沿径向方向由 中心向壁面先逐渐上升后降低,其中局部峰值的位 置随着床层高度的增加逐渐向壁面方向移动。其中 采用 Arastoopour模型和 DiFelice模型得到颗粒浓 度标准方差在中心射流区内较大,这是由于在床层 中产生了汽泡,从而导致了在中心射流和两侧环形 区的界面处颗粒浓度的标准方差值也较大。采用 S^yam lalOB rien模型计算时由于受到锥体影响较 大,因此在中心射流区和环形区的界面处颗粒浓度 方差值较高。从图中可以看出,采用 Gidaspow曳力 模型得到的颗粒浓度标准方差要小干采用 Arastoo pou模型、Syam la lO' Brien模型以及 Di Felice模型 得到的颗粒浓度标准方差值。

图 3表示入口射流速度 30 m/采用不同曳力 模型时反应器内颗粒浓度概率密度分布。模拟计算 结果表明在经过约 10 时间后流动进入稳定状态。 取此时间后的计算结果,对反应器内空间进行统计, 可以得到反应器内的浓度概率: $P(\varepsilon_s) = \sum_{i=1}^{l} \sum_{k=1}^{N} \frac{n_i}{N}$ 其 中 η 为颗粒浓度在 $\varepsilon_s \cong \varepsilon_s + \Delta \varepsilon_s$ 之间出现的次数, N为总统计数。显然, $\sum P(\varepsilon_i) = 1.0$ 针对不同曳 力模型的模拟结果表明,颗粒浓度的概率分布出现 了双峰分布的特征,分别是在低颗粒浓度区域和高 颗粒浓度区域。研究结果表明,靠近高颗粒浓度的 峰值贡献来自于两侧环形区内的稠密区颗粒流动。 而靠近低颗粒浓度的峰值贡献来自于中心射流区域 内的稀相区颗粒流动。两个峰值的分布取决于环形 区和中心射流区的界面流动过程。结合瞬时颗粒浓 度分布图可知,在中心射流区域内高速气体射流携 带向上运动,此区域是低颗粒浓度区出现概率高的 区域,形成了峰值。由于气体射流在向床面运动过 程中不断与射流区进行动量交换,极大消耗了射流 初始动量,直至到达最大床层膨胀高度时,由于气体 射流速度的骤然降低,颗粒因重力作用沿两侧环形 区内回流。因此在环形区内是出现高颗粒浓度区概 率较高的区域。



图 3 颗粒浓度的概率分布







图 5 颗粒浓度的概率分布

3 模拟结果和实验结果的比较

迄今,喷动床颗粒浓度测量数据十分有限,大部 分采用了对于稠相测量并不敏感的光纤技术。由于 直径为 25 mm的 D类颗粒流化床实验结果尚未见, 因此采用李水清等实验条件结合本研究的理论模型 进行预测。其中李水清等应用电容探针技术对喷动 床颗粒浓度进行了测量^[10]。实验装置为内径 192 mm,高 1 200 mm的圆柱体和锥角 60°、高 144 mm 的圆锥体连接而成的喷动床,入口直径为 25.72 mm。假设喷动床初始条件为临界流化状态。临界 流化速度和流化状态空隙率根据 Ergun方程迭代计 算得出。射流入口速度设定为给定值,出口采用压 力出口条件。

图 4和图 5分别表示表观气体速度为 0 460 m/时喷动床内颗粒浓度分布和颗粒浓度的概率分 布。由颗粒浓度沿径向方向分布图可知,在中心喷 射区处的颗粒浓度较高。随着径向距离的增大,颗 粒浓度逐渐降低到局部最小值后逐渐上升。在环隙 区域内颗粒,浓度基本保持不变。当采用。GidasPow 曳力模型得到的在床高 h=190 mm颗粒浓度沿径 向方向分布与李水清等人采用电容探针技术得到的 测量值趋势比较接近^[10]。其中采用 Arastoopour和 DiFelice曳力模型得到的颗粒浓度分布与实验测量 值误差较大。由图可知,在中心射流区域内颗粒浓 度大于测量值,而在靠近壁面区域的颗粒浓度值要 小于测量值。从颗粒浓度概率分布图可以看出,颗 粒浓度的概率分布分别在低颗粒浓度区域内出现概 率较小的峰值和高颗粒浓度区域内概率较大的峰 值。研究结果表明,靠近低颗粒浓度区域的峰值主 要是由中心射流区域内稀相区的颗粒流动引起的; 在高浓度区域的峰值是由两侧环形区内稠密区颗粒 流动引起的。

4 颗粒弹性恢复系数的影响

图 6是入口射流速度 30 m/ \$弹性恢复系数分 别为 =0 8 =0 9 =0 95时床内颗粒浓度的概 率分布。从图中可以看出,当颗粒弹性恢复系数较 小时,在颗粒浓度为 0 6时,存在较大的颗粒浓度概 率分布峰值;弹性恢复系数为 0.9时,在颗粒浓度为 0 6则颗粒浓度概率峰值是逐渐降低的,在低颗粒 区域的颗粒浓度概率峰值是逐渐增加的;当弹性恢 复系数增加到 0 95时,在低颗粒区域的颗粒浓度概 率分布出现了较大的峰值,而在高颗粒浓度区域的 颗粒浓度概率峰值逐渐消失,研究结果表明,流场内 颗粒浓度分布趋于均匀。

5 结 论

采用流体与颗粒两相流理论考虑两相间作用, 对流化床核反应器内流体动力特性进行了数值 模拟。

(1) 曳力是流化床核反应器内颗粒所受的主要 加速力, 曳力模型的选择对反应器内颗粒浓度的分 布有重要的影响。采用 Gidaspow曳力模型得到的 颗粒浓度分布比较均匀; 使采用流态化技术的流化 床核反应器内燃料颗粒浓度的分布变得均匀, 提高 了核反应器的传热能力。

(2)采用颗粒浓度的概率分布来描述流场内颗 粒分布的整场特征,研究发现增加颗粒碰撞恢复系 数使得颗粒浓度概率分布在高浓度区的峰值逐渐降 低,引起流场内颗粒浓度分布变得均匀,有利于提高 核反应器的安全性。reserved. http://www.cnki.net



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图 6 颗粒浓度的概率分布

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(编辑 陈 滨)



With Fluent₆ 3 serving as a platform, a partial heat non equilibrium model was adopted to numerically study the non Darcian forced convection heat exchange in the vertical passages of a skeleton heat generation porous medium in a turbulent flow zone and its transition one. Three dimensional N—S equation and standard k— ε turbulent flow model were used to depict the flow inside the porous medium. On this basis, the influence of the change in the pore effective Reynolds number Re (400< R< 2000), surface heat flux density q(Φ =5 kW/m², 30 kW/m² and 90 kW/m²) and cookint inlet temperature T_n (T_n=20 °C, 50 °C and 80 °C) on the flow resistance and heat exchange characteristics was studied in detail. The research results show that at a low heat flux density the change of surface heat flux density has a very small influence on the flow resistance and heat exchange coefficient. However, the diameter of the small balls exercises a significant influence on the heat exchange coefficient and such an influence with increase with an increase of Reynolds number. Moreover, the heat exchange coefficient will decrease with an increase of the cookint inlet temperature Keywords skeleton heat generation, porous medium, numerical simulation

膜式全热换热器 EHD电场强化换热的实验研究 = Experimental Study of the EHD(Electrohydrodynamics)-based Electric Field Intensified Heat Exchange of a Mem brane Type Full Heat Exchangen 刊,汉]/ SUN Shuhong IU Yuan wei LU Guang lin et al Education Ministry Key Laboratory on Heat Transfer Intensifi cation and Process Energy Conservation, Beijing University of Technology Beijing China PostCode 100124)// Journal of Engineering for Thermal Energy & Power - 2010 25(6). -617~620

To enhance the heat exchange efficiency of a membrane type full heat exchanger a high voltage electric field was applied to the heat exchanger. Under the same test conditions the influence of the electric field applied from out side on the heat exchange effectiveness was analyzed by measuring both sensible and latent heat efficiency of the exchanger. On this basis, the heat exchange effectiveness of the exchanger was tested at various voltages of electric poles and different wind speeds. The test results show that the application of a high voltage electric field to the flow field of the heat exchanger can effectively enhance its sensible heat efficiency but insignificantly increase its latent heat efficiency. At a low wind speed, the intensified heat exchange effectiveness will be even more conspicuous K ey words full heat exchanger intensified heat exchange electrohyd rodynamics (EHD)

流化床反应器内气固两相流动特性的研究 = Investigation of the Gassolid Two phase Flow Character istics Inside a Fluid zed Bed R eacto [刊,汉] / SUN Qiaoqu, ZHUW eibing (College of Astronautics and Architec tural Engineering Hathin Engineering University Hathin, China, PostCode, 150001), GAO Jianmin, IUHui lin (College of Energy Science and Engineering, Hathin Institute of Technology, Hathin, China, Post Code 150001)// Journal of Engineering for Thermal Energy & Power - 2010, 25(6). -621~626

By simulating the particle phase flow based on the particle kinetic theory and taking into account the two phase in teraction by using the fluid and particle two phase flow theory established was a CFD (computational fluid dynamics) model featuring the multiple phase flow inside a fluidized bed nuclear reactor and numerically simulated and studied were the fluid kinetic behaviors in the above mentioned reactor. The calculation results show that the distribution of particle concentrations on the cross section obtained by using G idaspow drag force model shares a compar ?1994-2018 China Academic Journal Electronic Publishing House. All rights reserved. http://www.cnki.net

atively close tendency to that of the currently available test results. The particle concentration at the center of the central jet zone is relatively high. After dropping step by step to a partial minimum value, it will gradually increase with an increase of the radial distance. The particle concentration in the annulus zone remains basically unchanged. The influence of the action force between the fluid and particles as well as the particle elastic recovery coefficient etc. on the kinetic characteristics of the fluid in the reactor was analyzed. The research results show that the greater the particle collision recovery coefficient themore uniform the particle concentration distribution on the cross section in the flow field. Key words fluid zed bed nuclear reactor, two phase flow theory numerical simulation

循环流化床锅炉热效率统计分析研究 = Statistical Analysis and Study of the Thermal Efficiencies of a Circulating Fluidized Bed Boile町刊,汉] / JANG Shao jian, LU Le AI Yuan fang (College of Energy Science and Engineering Central South University Changsha China PostCode 410083), HE Xiang zhu (Hunan Provincial Energy Conservation Center Changsha, China PostCode 410007) // Journal of Engineering for Thermal Energy & Power - 2010 25(6). -627~629

In the light of the problem that the adoption of the empirical comparison method has a poor adaptability to the fur nace volume of a CFB boiler a power function regularity was used to perform a fitting of the operating data of a CFB boiler. On this basis, the relationship between the thermal efficiency and the main influencing factors (such as ton steam effective volume volatile content of the coal) of the boiler was studied and a concept of ton steam effective volume, put forward. The research results show that the tonnage steam effective volume and the volatile content of the coal burned are the major factors influencing the furnace type selection. Tom ake the thermal efficiency of the boiler attain over 80%, the tonnage steam effective volume (represented by letter y) and the volatile content of the coal (represented by letter x) shall meet the requirement below \gg 7. 78 x^{0 136}. Key words circulating fluid zed bed boiler furnace volume volatile content regression analysis thermal efficiency on steam effective volume

炉内燃烧器射流组组合特性分析 = Analysis of the JetF bw Group Combination Characteristics of a In fur nace Burner[刊,汉] / SHIGuang-me, LIM ing ha, CHEN Jun et al (Structural Mechanics Research Institute Chinese Academy of Engineering Physics Mianyang China Post Code 621900) // Journal of Engineering for The mal Energy & Power - 2010 25(6). -630~634

With an oil fired boiler serving as a concrete object of study the infumace three dimensional turbulent flow combustion field characteristics were numerically simulated by adjusting several main combination modes of the burners and the regularity of the jet flow group combination characteristics of the burners influencing the infumace aerody namic field was obtained. A comparison of the simulation calculated results with the test ones shows that the calculated data are in relatively good agreement with the actually measured ones. This is of realistic significance for determining the combined operating condition of the burners and designing a test scheme, thereby providing a theoret ical reference and basis for regulating the operating condition of indoor oil fired boilers. Key words turbulent flow combustion jet flow group combination characteristics numerical simulation oil fired boiler